



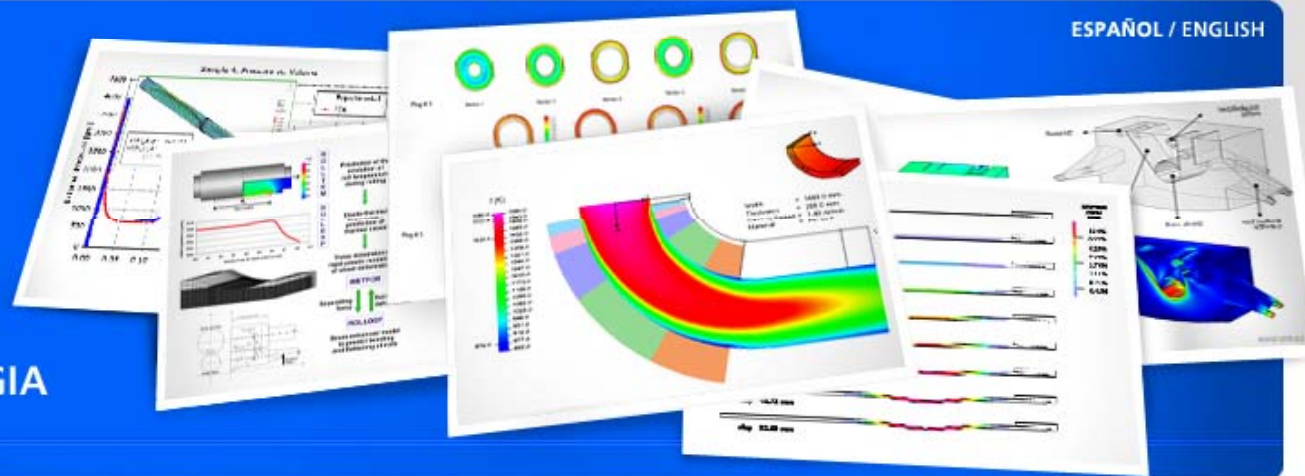
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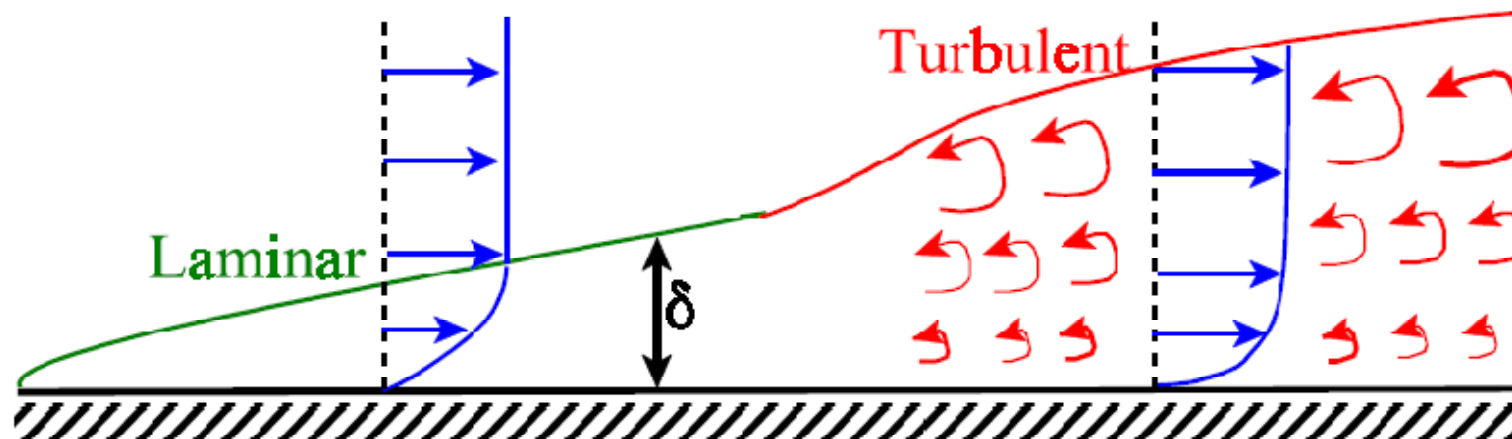


FINITE ELEMENT METHOD IN FLUID DYNAMICS

Part 1: Laminar flow

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Laminar Flow



Laminar Flow

Ideal Fluids

- ✓ Fluid with no friction
- ✓ Also referred to as an *inviscid* (zero viscosity) fluid
- ✓ Internal forces at any section within are normal (pressure forces)
- ✓ Practical applications: many flows approximate frictionless flow away from solid boundaries.
- ✓ Do not confuse ideal fluid with a perfect (ideal) gas.

Laminar Flow

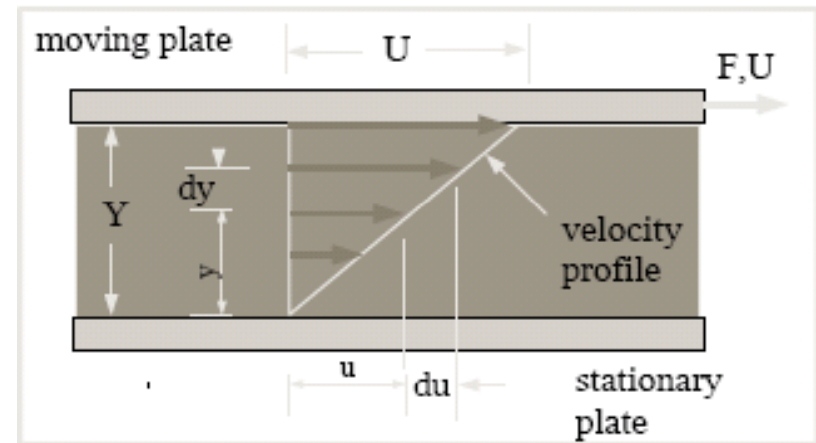
Real Fluids

- ✓ Tangential or shearing forces always develop where there is motion relative to solid body
- ✓ Shear forces oppose motion of one particle past another
- ✓ Friction forces gives rise to a fluid property called viscosity

Laminar Flow

Viscosity

- A measure of a fluid's resistance to angular deformation, e.g.,
- Motor oil: high viscosity, feels sticky
 - Gasoline: low viscosity, flows “faster”



Friction forces result from cohesion and momentum interchange between molecules.

The shear stress between layers $\tau = \mu \frac{du}{dx}$

μ = coefficient of viscosity, absolute viscosity, dynamic viscosity, or simply viscosity

Laminar Flow

Viscosity

- In B.G. Units

$$[\mu] = \frac{[\tau]}{[du/dy]} = \frac{lb/ft^2}{fps/ft} = \frac{lb \cdot sec}{ft^2}$$

- In S.I. Units

$$[\mu] = \frac{[\tau]}{[du/dy]} = \frac{N/m^2}{s^{-1}} = \frac{N \cdot s}{m^2}$$

- The *poise* (P):
 - Metric unit of viscosity
 - Named after Jean Louis Poiseuille (1799-1869)
 - The poise: $1 P = 0.10 N \cdot s/m^2$
 - The *centipoise*: $1 cP = 0.01 P = 1 mN \cdot s/m^2$
 - For water at $68.4^\circ F$ ($20.22^\circ C$), $\mu = 1 cP$

Laminar Flow

Viscosity

- Ratio of absolute viscosity to density
- Appears in many problems in fluids
- Called *kinematic viscosity* because it involves no force (dynamic) dimensions
- B.G. Units = ft^2/sec , S.I. Units = m^2/s
- The *stoke* (St)
 - Metric unit of kinematic viscosity
 - Named after Sir George Stokes (1819-1903)
 - The *centistoke*: $1 cSt = 0.01 St = 10^{-6} m^2/s$

$$\nu = \frac{\mu}{\rho}$$

μ for most fluids is virtually independent of pressure for the range of interest to engineers

ν for gases varies strongly with pressure because of changes in density (ρ)

Laminar Flow

Viscosity

| Liquid | Viscosity in mPa.s |
|-------------------|--------------------|
| Water at 0°C | 1.79 |
| Water at 20°C | 1.002 |
| Water at 100°C | 0.28 |
| Glycerin at 0°C | 12070 |
| Glycerin at 20°C | 1410 |
| Glycerin at 30°C | 612 |
| Glycerin at 100°C | 14.8 |
| Mercury at 20°C | 1.55 |
| Mercury at 100°C | 1.27 |
| Motor Oil SAE 30 | 200 |
| Motor Oil SAE 60 | 1000 |
| Ketchup | 50,000 |

| Gas | Viscosity in 10 ⁻⁶ Pa.s |
|------------------|------------------------------------|
| Air at 100K | 7.1 |
| Air at 300K | 18.6 |
| Air at 400K | 23.1 |
| Hydrogen at 300K | 9.0 |
| Helium at 300K | 20.0 |
| Oxygen at 300K | 20.8 |
| Nitrogen at 300K | 17.9 |
| Xenon at 300K | 23.2 |

Laminar Flow

Density

The **density** of a material is defined as its mass per unit volume

$$\begin{aligned}\rho(x, y, z) &= \lim_{Volume \rightarrow 0} \frac{\text{mass of box}}{\text{volume of box}} \\ &= \lim_{\Delta x, \Delta y, \Delta z \rightarrow 0} \left(\frac{m(x + \Delta x, y + \Delta y, z + \Delta z) - m(x, y, z)}{\Delta x \Delta y \Delta z} \right) \\ &= \frac{dm}{dV}.\end{aligned}$$



Laminar Flow

Density

| Material | ρ in kg/m ³ | Notes |
|----------|-----------------------------|------------------------|
| Water | 1000 | At STP |
| Iron | 7874 | Near room temperature |
| Copper | 8920 – 8960 | Near room temperature |
| Lead | 11340 | Near room temperature |
| Gold | 19300 | Near room temperature |
| Platinum | 21450 | Near room temperature |
| Air | 1.184 | Near room temperature |

Laminar Flow

Mass conservation equation

“Mass cannot disappear or created from the system”

$$\frac{D\rho}{Dt} + \rho \frac{\partial v_i}{\partial x_i} = 0 \qquad \frac{\partial \rho}{\partial t} + v_i \frac{\partial \rho}{\partial x_i} + \rho \frac{\partial v_i}{\partial x_i} = 0$$

Incompressible flow

$$\frac{\partial v_i}{\partial x_i} = 0 \quad \rightarrow \quad \frac{\partial v_1}{\partial x_1} + \frac{\partial v_2}{\partial x_2} + \frac{\partial v_3}{\partial x_3} = 0 \quad \rightarrow \quad \nabla \cdot \underline{v} = 0$$

- ▶ The incompressibility condition does not imply density uniform ($\partial\rho/\partial x_i=0$) or constant ($\partial\rho/\partial t=0$). Flow of two fluids of different density.
- ▶ A flow can be consider incompressible when the fluid particle velocities are lesser than the sonid velocity (Mach number $\ll 1$).

Laminar Flow

Mass conservation equation

Cartesian coordinates

$$\frac{\partial \rho}{\partial t} + \frac{\partial}{\partial x}(\rho v_x) + \frac{\partial}{\partial y}(\rho v_y) + \frac{\partial}{\partial z}(\rho v_z) = 0$$

Cylindrical coordinates

$$\frac{\partial \rho}{\partial t} + \frac{1}{r} \frac{\partial}{\partial r}(r \rho v_r) + \frac{1}{r} \frac{\partial(\rho v_\theta)}{\partial \theta} + \frac{\partial(\rho v_z)}{\partial z} = 0$$

Laminar Flow

Momentum Conservation

$$\rho \frac{D v_i}{D t} = \frac{\partial \sigma_{ij}}{\partial x_j} + \rho f_i$$

Newton's Second Law: “ The momentum principle for a collection of particles states that the time rate of change of the total momentum of a given set of particles equals the vector sum of all the external forces acting on the particles of the set provided Newton's Third Law of action and reaction governs the internal forces. The continuum form of this principle is a basic postulate of continuum mechanics”

Laminar Flow

Momentum Conservation

The ***Newtonian model*** of fluid response is based on three assumptions:

- (a) shear stress is proportional to the *rate* of shear strain in a fluid particle;
- (b) shear stress is zero when the rate of shear strain is zero;
- (c) the stress to rate-of-strain relation is isotropic—that is, there is no preferred orientation in the fluid.

A Newtonian fluid is the simplest type of viscous fluid, just like an elastic solid (where stresses are proportional to strains) is the simplest type of deformable solid.

Laminar Flow

Momentum Conservation

$$\sigma_{ij} = - \left(p + \frac{2}{3} \mu \nabla \cdot \underline{v} \right) \delta_{ij} + 2 \mu S_{ij}$$

$$S_{ij} = \frac{1}{2} \left(\frac{\partial v_i}{\partial x_j} + \frac{\partial v_j}{\partial x_i} \right)$$

For incompressible flow

$$\rho \frac{\partial \underline{v}}{\partial t} + \rho \underline{v} \cdot \nabla \underline{v} = -\nabla p + \nabla \cdot \left[\mu \left(\nabla \underline{v} + \nabla \underline{v}^T \right) \right] + \rho \underline{f}$$

Laminar Flow

Momentum Conservation

For incompressible flow and constant viscosity

$$\rho \frac{\partial \underline{v}}{\partial t} + \rho \underline{v} \cdot \nabla \underline{v} = -\nabla p + \mu \nabla^2 \underline{v} + \rho \underline{f}$$

Cartesian coordinates

$$\rho \left(\frac{\partial v_x}{\partial t} + v_x \frac{\partial v_x}{\partial x} + v_y \frac{\partial v_x}{\partial y} + v_z \frac{\partial v_x}{\partial z} \right) = - \frac{\partial p}{\partial x} + \mu \left(\frac{\partial^2 v_x}{\partial x^2} + \frac{\partial^2 v_x}{\partial y^2} + \frac{\partial^2 v_x}{\partial z^2} \right) + \rho f_x$$

$$\rho \left(\frac{\partial v_y}{\partial t} + v_x \frac{\partial v_y}{\partial x} + v_y \frac{\partial v_y}{\partial y} + v_z \frac{\partial v_y}{\partial z} \right) = - \frac{\partial p}{\partial y} + \mu \left(\frac{\partial^2 v_y}{\partial x^2} + \frac{\partial^2 v_y}{\partial y^2} + \frac{\partial^2 v_y}{\partial z^2} \right) + \rho f_y$$

$$\rho \left(\frac{\partial v_z}{\partial t} + v_x \frac{\partial v_z}{\partial x} + v_y \frac{\partial v_z}{\partial y} + v_z \frac{\partial v_z}{\partial z} \right) = - \frac{\partial p}{\partial z} + \mu \left(\frac{\partial^2 v_z}{\partial x^2} + \frac{\partial^2 v_z}{\partial y^2} + \frac{\partial^2 v_z}{\partial z^2} \right) + \rho f_z$$

Cylindrical coordinates

$$\rho \left(\frac{\partial v_r}{\partial t} + v_r \frac{\partial v_r}{\partial r} + \frac{v_\theta}{r} \frac{\partial v_r}{\partial \theta} - \frac{v_\theta^2}{r} + v_z \frac{\partial v_r}{\partial z} \right) =$$

$$-\frac{\partial p}{\partial r} + \mu \left[\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_r}{\partial r} \right) - \frac{v_r}{r^2} + \frac{1}{r^2} \frac{\partial^2 v_r}{\partial \theta^2} - \frac{2}{r^2} \frac{\partial v_\theta}{\partial \theta} + \frac{\partial^2 v_r}{\partial z^2} \right] + \rho f_r$$

$$\rho \left(\frac{\partial v_\theta}{\partial t} + v_r \frac{\partial v_\theta}{\partial r} + \frac{v_\theta}{r} \frac{\partial v_\theta}{\partial \theta} + \frac{v_r v_\theta}{r} + v_z \frac{\partial v_\theta}{\partial z} \right) =$$

$$-\frac{1}{r} \frac{\partial p}{\partial \theta} + \mu \left[\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_\theta}{\partial r} \right) - \frac{v_\theta}{r^2} + \frac{1}{r^2} \frac{\partial^2 v_\theta}{\partial \theta^2} + \frac{2}{r^2} \frac{\partial v_r}{\partial \theta} + \frac{\partial^2 v_\theta}{\partial z^2} \right] + \rho f_\theta$$

$$\rho \left(\frac{\partial v_z}{\partial t} + v_r \frac{\partial v_z}{\partial r} + \frac{v_\theta}{r} \frac{\partial v_z}{\partial \theta} + v_z \frac{\partial v_z}{\partial z} \right) =$$

$$-\frac{\partial p}{\partial z} + \mu \left[\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 v_z}{\partial \theta^2} + \frac{\partial^2 v_z}{\partial z^2} \right] + \rho f_z$$

Laminar Flow

Momentum Conservation

The pressure does not have an evolution equation. In each time instant, the pressure is adjusted to fulfill the incompressibility condition.

The viscosity is a fluid property, depend only to the temperature for Newtonian fluids.

Laminar Flow

Non-dimensional numbers

$$\text{Re} = \frac{\rho V L}{\mu} \quad \frac{\textit{inertial forces}}{\textit{viscous forces}}$$

$$\text{Fr} = \frac{V^2}{g L} \quad \frac{\textit{inertial forces}}{\textit{gravitatio nal forces}}$$

$$\text{Ma} = \frac{V}{c} \quad \frac{\textit{fluid speed}}{\textit{speed of the sound in the fluid}}$$

Laminar Flow

Non-dimensional numbers

$$Eu = \frac{\Delta P}{\frac{1}{2} \rho V^2} \quad \frac{\text{pressure forces}}{\text{inertial forces}}$$

$$We = \frac{V}{\sqrt{\frac{\sigma}{\rho L}}} \quad \frac{\text{inertial forces}}{\text{surface tension}}$$

Laminar Flow

Boundary Conditions

In the absence of surface tension, the boundary conditions consistent with the continuum hypothesis are that

(a) the velocity components

(b) the stress tensor components must be everywhere continuous,

including across phase interfaces like the boundaries between the fluid and a solid and between two immiscible fluids.

Laminar Flow

Boundary Conditions

- ▶ Essential BC: velocities imposed

$$v_i = w_{imp}$$

- ▶ Natural BC: traction imposed

$$\underline{\underline{\sigma}} \cdot \underline{n} = \underline{t} \quad \rightarrow \quad \mu \nabla \underline{v} \cdot \underline{n} - p \underline{n} = \underline{t}$$

Laminar Flow

Energy Conservation

$$\underbrace{\rho C_p \frac{\partial T}{\partial t}}_{\text{Transient term}} + \underbrace{\rho C_p \underline{v} \cdot \underline{\nabla} T}_{\text{Convective term}} = \underbrace{\underline{\nabla} \cdot (\underline{k} \cdot \underline{\nabla} T)}_{\text{Diffusive term}} + \underbrace{q_v}_{\text{Volumetric heat [W/m}^3\text{]}}$$

ρ density [Kg/m³]

C_p Specific heat [J/Kg °K]

\underline{k} Thermal conductivity [W/M°K]

\underline{v} Velocity [m/s]

$$\underline{q} = -k \underline{\nabla} T$$

\underline{q} : heat flux [W/m²]

Laminar Flow

Specific heat capacity

| Substance | Phase | C_p J/(g·K) | $C_{p,m}$ J/(mol·K) | $C_{v,m}$ J mol ⁻¹ ·K ⁻¹ | Volumetric heat capacity J/(cm ³ ·K) |
|---|--------|------------------|------------------------|---|---|
| Air (Sea level, dry, 0 °C) | gas | 1.0035 | 29.07 | 20.7643 | 0.001297 |
| Air (typical room conditions ^A) | gas | 1.012 | 29.19 | 20.85 | |
| Aluminium | solid | 0.897 | 24.2 | | 2.422 |
| Carbon dioxide CO ₂ | gas | 0.839* | 36.94 | 28.46 | |
| Copper | solid | 0.385 | 24.47 | | 3.45 |
| Ethanol | liquid | 2.44 | 112 | | 1.925 |
| Gasoline | liquid | 2.22 | 228 | | 1.64 |
| Glass | solid | 0.84 | | | |
| Gold | solid | 0.129 | 25.42 | | 2.492 |
| Iron | solid | 0.450 | 25.1 | | 3.537 |
| Lead | solid | 0.127 | 26.4 | | 1.44 |
| Mercury | liquid | 0.1395 | 27.98 | | 1.888 |
| Methane at 2 °C | gas | 2.191 | | | |
| Water at 100 °C (steam) | gas | 2.080 | 37.47 | 28.03 | |
| Water at 25 °C | liquid | 4.1813 | 75.327 | 74.53 | 4.186 |
| Water at -10 °C (ice) | solid | 2.05 | 38.09 | | 1.938 |

Laminar Flow

Thermal conductivity

| Material | Thermal conductivity [W/(m·K)] |
|-----------------|--------------------------------|
| Air | 0.025 |
| Water (liquid) | 0.6 |
| Glass | 1.1 |
| Soil | 1.5 |
| Concrete stone | 1.7 |
| Ice | 2 |
| Sandstone | 2.4 |
| Stainless steel | 12.11 ~ 45.0 |
| Lead | 35.3 |
| Aluminium | 237 (pure) 120—180 (alloys) |
| Gold | 318 |
| Copper | 401 |
| Silver | 429 |
| Diamond | 900 - 2320 |
| Graphene | (4840±440) - (5300±480) |

Laminar Flow

Energy Conservation – Non dimensional numbers

$$Gr = \frac{g \beta (T_s - T_\infty) L^3}{V^2} \quad \frac{\text{bouyancy forces}}{\text{viscous forces}}$$

β = volumetric thermal expansion coefficient (equal to approximately $1/T$, for ideal fluids, where T is absolute temperature)

$$Pr = \frac{C_p \mu}{k} \quad \frac{\text{viscous diffusion rate}}{\text{thermal diffusion rate}}$$

$$Br = \frac{\mu V^2}{k (T_{wall} - T_{bulk})} \quad \frac{\text{heat production by viscous dissipation}}{\text{heat production by conduction}}$$

Laminar Flow

Energy Conservation - Non dimensional numbers

$$Ec = \frac{Br}{Pr} = \frac{V^2}{Cp \Delta T} \quad \frac{\text{Kinetic energy}}{\text{Enthalpy}}$$

$$Nu = \frac{h L}{k_f} \quad \frac{\text{convective heat transfer}}{\text{conductive heat transfer}}$$